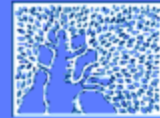




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## 2<sup>nd</sup> International Conference and Advanced School “Turbulent Mixing and Beyond”

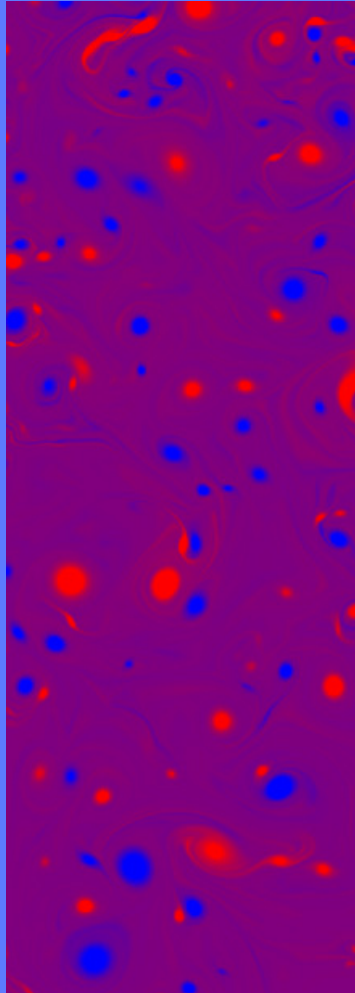
# Velocity and energy profiles in two- vs. three-dimensional channels: Effects of an inverse vs. a direct energy cascade

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Rehovot, Israel*



# Overview



## **Formulation of the Problem**

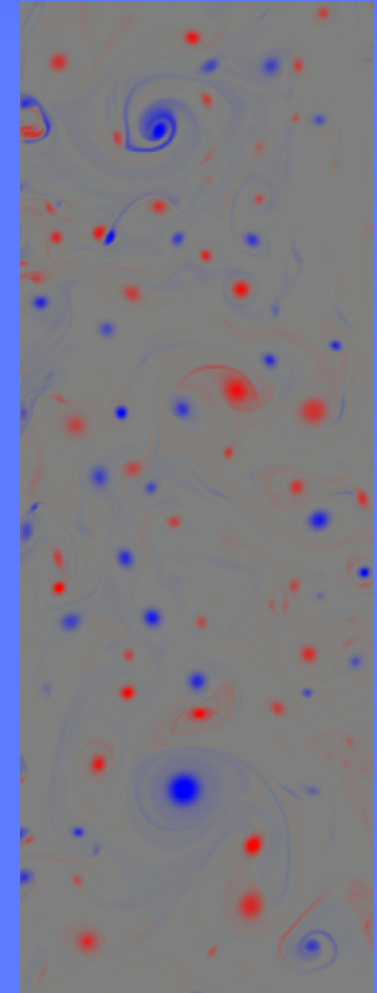
### **Milestones of the Model**

- Mean Momentum Balance
- Kinetic Energy Balance
- Reynolds Stress Balance
- “Outer scale” of turbulence

### **Results**

- Mean Velocity Profiles
- Kinetic Energy Profiles
- Reynolds Stress Profiles
- Kinetic Energy Balance

### **Conclusions**

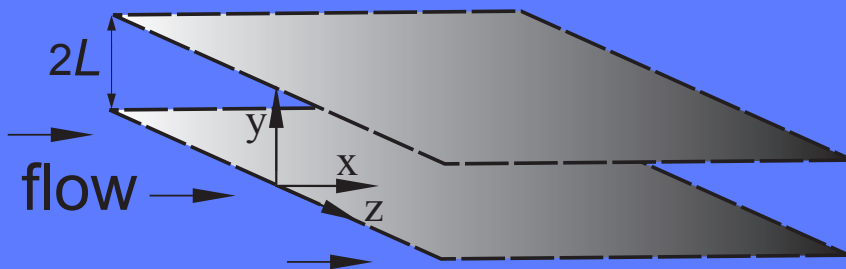




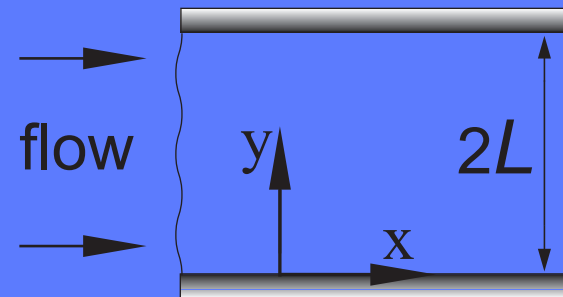
# Problem Formulation

Stationary fully developed turbulent flow

**3D Channel**



**2D Channel**



For 3D:

$z$  – spanwise direction,  
 $-\infty \leq z \leq \infty$ .

$x$  – streamwise direction,  
 $-\infty \leq x \leq \infty$ ,

$y$  – wall-normal direction,  
 $0 \leq y \leq 2L$ .

Driven force – constant pressure gradient:  $p' \equiv -d \langle p(x) \rangle / dx > 0$ .

Fluid velocity (Reynolds decomposition):  $\mathbf{U}(\mathbf{r}) = V(y) \hat{\mathbf{x}} + \mathbf{u}(\mathbf{r})$ ,  $V(y) \equiv \langle \mathbf{U}(\mathbf{r}) \rangle$ .



# Mean Momentum balance

$$\nu S(y) + W(y) = p'(L - y).$$

Mean Shear:  $S(y) \equiv \frac{dV(y)}{dy},$

Reynolds Shear Stress:  $W(y) \equiv -\langle u_x u_y \rangle,$

Turbulent Kinetic Energy:  $K(y) \equiv \frac{1}{2} \langle \mathbf{u}^2 \rangle.$



# Kinetic Energy Balance

$$P(y) = \varepsilon(y) + D(y),$$

Energy Production:  $P(y) = W(y)S(y),$

Energy Dissipation:  $\varepsilon(y) = \nu \left\langle \left( \partial_k u_i \right)^2 \right\rangle,$

Energy Diffusion:  $D(y) = \frac{d}{dy} \left[ \frac{1}{2} \left\langle u_y \left( \mathbf{u}^2 + \tilde{p} \right) \right\rangle - \nu \frac{d}{dy} K(y) \right].$

**Model** for Diffusion:

$$D(y) \approx \frac{d}{dy} \left\{ \left[ -\nu_T(y) \frac{d}{dy} K(y) \right] - \nu \frac{d}{dy} K(y) \right\},$$

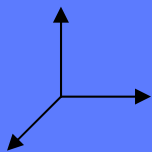
$$\nu_T(y) = a \ell(y) \sqrt{K(y)} \quad .$$



# Dissipation in the bulk

Dissipation:  $\varepsilon(y) \approx \nu \int dk k^2 \tilde{K}(k),$

Kinetic energy:  $K(y) \approx \int_{1/\ell(y)}^{\infty} dk \tilde{K}(k).$



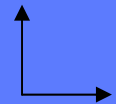
**3D Channel**

Direct *energy* cascade, Kolmogorov spectrum

$$\tilde{K}_{3D} \sim \varepsilon^{2/3} k^{-5/3},$$

$$\varepsilon_{3D} \sim \frac{K^{3/2}}{\ell},$$

**2D Channel**



Direct *enstrophy* cascade, Kraichnan spectrum

$$\tilde{K}_{2D} \sim \beta^{2/3} k^{-3} \ln^{-1/3} [k \ell(y)],$$

$$\varepsilon_{2D} \sim \nu \frac{K}{\ell^2} \ln^{2/3} \left[ \nu^{-1} \ell \sqrt{K} + \text{const} \right],$$



# Dissipation near walls

Near wall expansion:  $\varepsilon(y) \xrightarrow{y \rightarrow 0} 2\nu K(y)/y^2,$

$$\varepsilon(y) \approx 2\nu \frac{K(y)}{\ell(y)^2}, \quad \ell(y) \xrightarrow{y \rightarrow 0} y.$$

Model for Dissipation:

$$\varepsilon(y) \approx \begin{cases} 2\nu \frac{K(y)}{\ell(y)^2} + b \frac{K^{3/2}(y)}{\ell(y)}, & 3D, \\ 2\nu \frac{K(y)}{\ell^2(y)} \ln^{2/3} \left[ \nu^{-1} \ell(y) \sqrt{K(y)} + e \right], & 2D. \end{cases}$$



# Reynolds Stress Balance

Boussinesq closure:

$$W(y) \approx \nu_T(y) S(y),$$

$$\nu_T(y) \sim \ell(y) \sqrt{K(y)}.$$

$$r_W(y) W(y) \approx c \ell(y) \sqrt{K(y)} S(y),$$

$$r_W(y) = \left[ 1 + (\ell_{\text{buf}}/y)^6 \right]^{1/6}.$$

By fit to (3D) **D**irect **N**umerical **S**imulations (DNS) data:

$$\ell_{\text{buf}}^+ \equiv \ell_{\text{buf}} \sqrt{p' L} / \nu \approx 43.$$

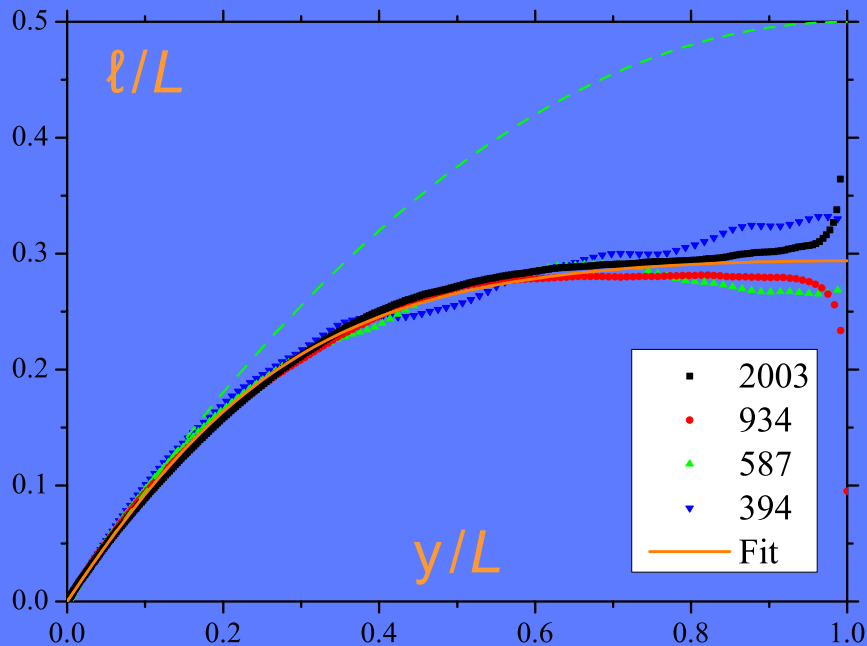
V. S. L'vov, I. Procaccia, and O. Rudenko, *Phys. Rev. Lett.* **100**, 054504 (2008).





# “Outer scale” of turbulence

“Extracted” from 3D channel and adopted for 2D channel.



$$L_S = 0.311 L ,$$

$$\lambda(y) = y(1 - y/2L)/L_S ,$$

$$\ell(y) = L_S \left\{ 1 - \exp \left[ -\lambda \left( 1 + \frac{\lambda}{2} \right) \right] \right\} .$$

Symbols: **D**irect **N**umerical **S**imulations (DNS) at four Reynolds numbers by

S. Hoyas and J. Jimenez, *Phys. Fluids* **18**, 011702 (2006);

R. G. Moser, J. Kim, and N. N. Mansour, *Phys. Fluids* **11**, 943 (1999);

The fit (orange solid line) is proposed in

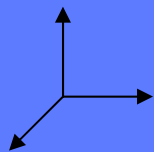
V. S. L’vov, I. Procaccia, and O. Rudenko, *Phys. Rev. Lett.* **100**, 054504 (2008).



# Final Set of Equations

$$\nu S + W = p'(L - y), \quad \left[1 + (\ell_{\text{buf}}/y)^6\right]^{1/6} W \approx c \ell \sqrt{K} S,$$

$$WS + \frac{d}{dy} \left[ (a \ell \sqrt{K} + \nu) \frac{d}{dy} K \right] \approx \begin{cases} 2\nu \frac{K}{\ell^2} + b \frac{K^{3/2}}{\ell}, & 3\text{D}, \\ 2\nu \frac{K}{\ell^2} \ln^{2/3} [\nu^{-1} \ell \sqrt{K} + e], & 2\text{D}. \end{cases}$$

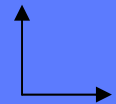


**3D Channel**

$$\kappa_{3\text{D}} = (c^3/b)^{1/4} \approx 0.415.$$

By fit to (3D) DNS data:

$$a \approx 0.218, \quad b \approx 0.310, \quad (c \approx 0.386).$$



**2D Channel**

$$\kappa_{2\text{D}} \approx 0.2, \quad \text{at } \text{Re}_\tau \sim 10^3.$$

N. Gutterberg and N. Goldenfeld,  
*Phys. Rev. E* **79**, 065306(R) (2009)

$$c \approx 0.047.$$

We fix the same values for 2D and 3D for  $a, \ell_{\text{buf}}, L_s$ .



# Mean Velocity Profiles

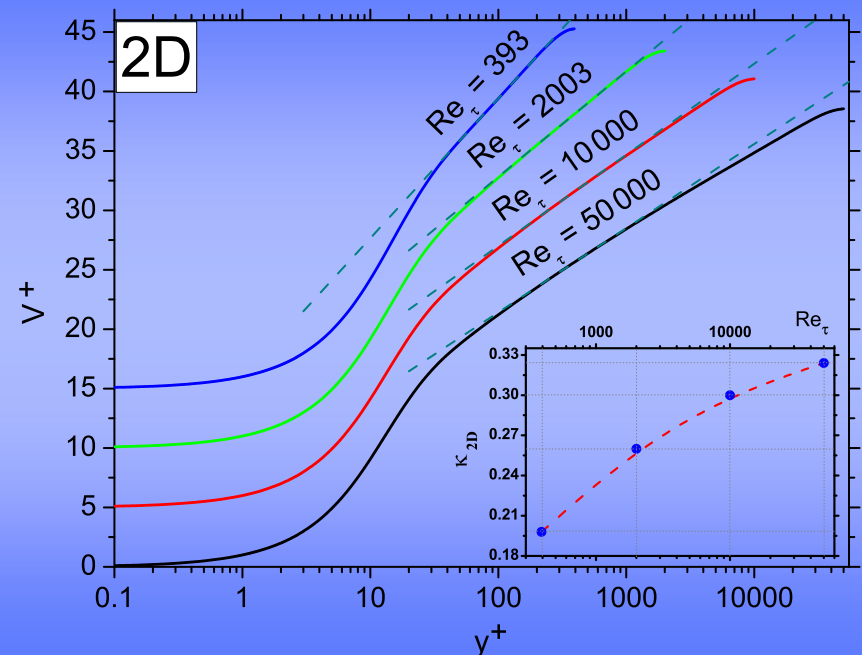
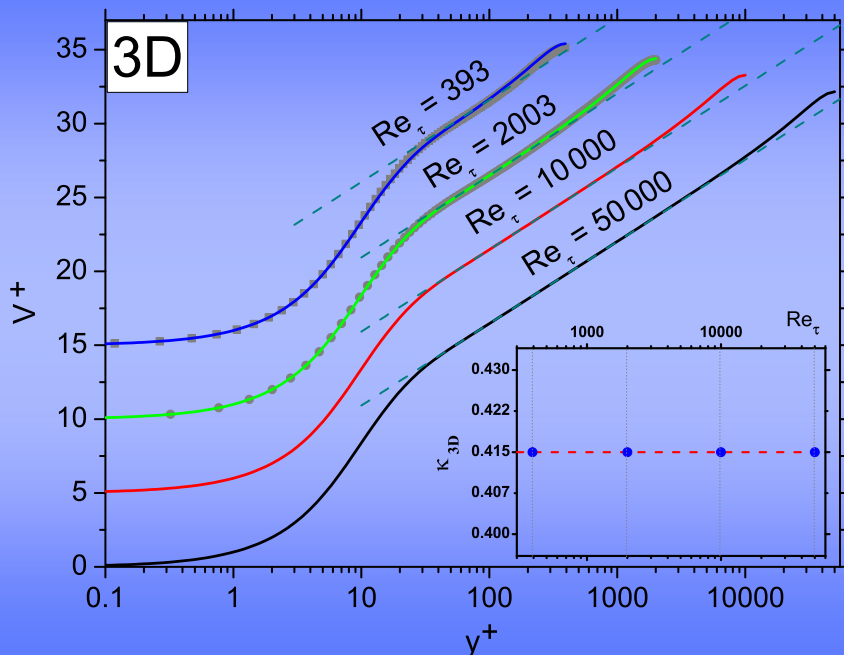
Wall Units:

$$u_\tau = \sqrt{p'L}, \quad \ell_\tau = \nu/u_\tau.$$

Friction Reynolds Number:

$$\text{Re}_\tau = Lu_\tau/\nu = L^+.$$

$$V^+ = V/u_\tau, \quad y^+ = y/\ell_\tau = y\nu/u_\tau.$$

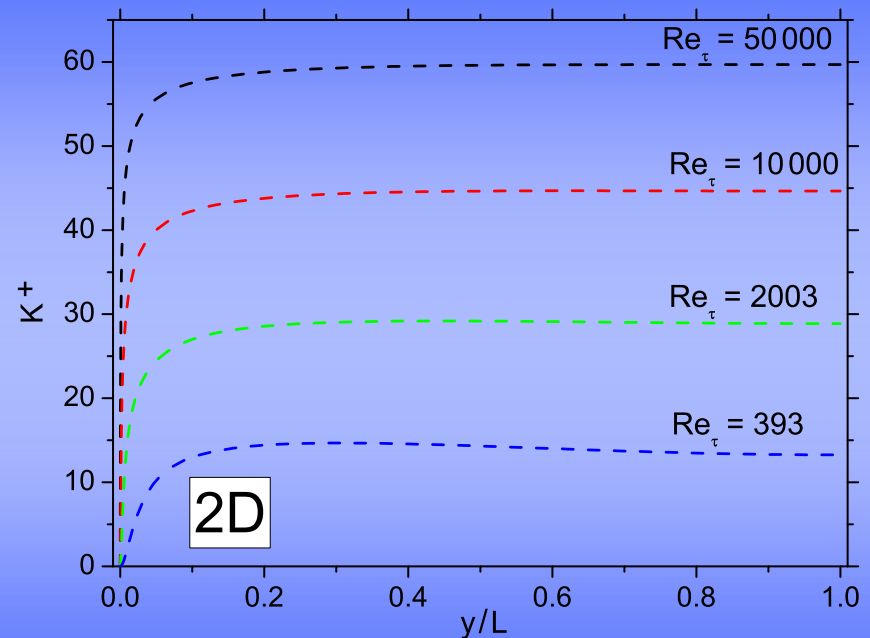
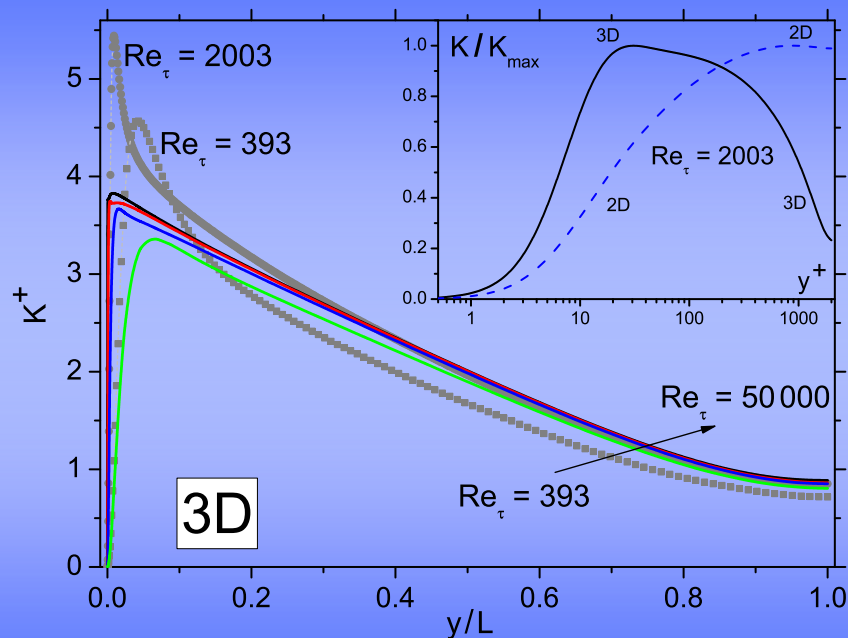


Symbols: Direct Numerical Simulations



# Kinetic Energy Profiles

$$K^+ = K/u_\tau^2, \quad Re_\tau = Lu_\tau/\nu.$$



Symbols: Direct Numerical Simulations by

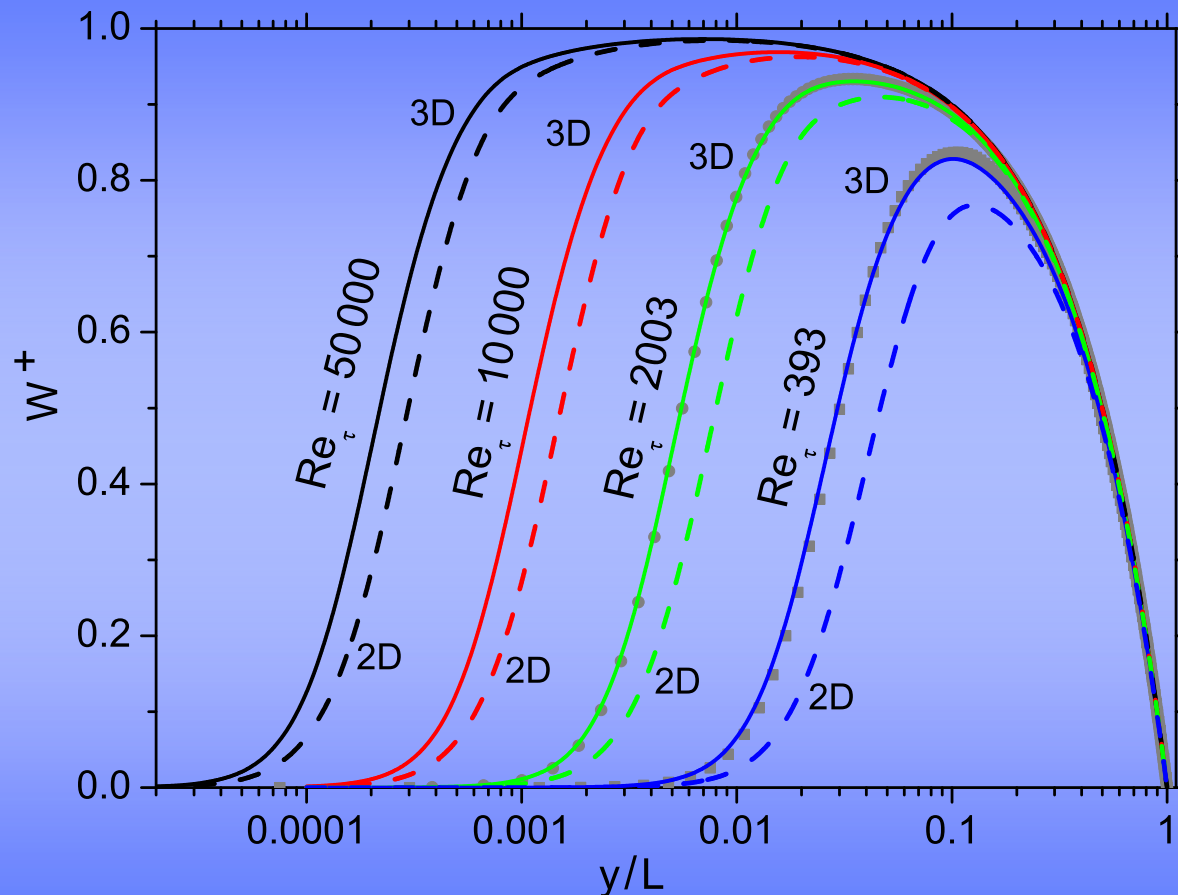
S. Hoyas and J. Jimenez, *Phys. Fluids* **18**, 011702 (2006);

R. G. Moser, J. Kim, and N. N. Mansour, *Phys. Fluids* **11**, 943 (1999).



# Reynolds Stress Profiles

$$W^+ = W/u_\tau^2, \quad Re_\tau = Lu_\tau/\nu.$$

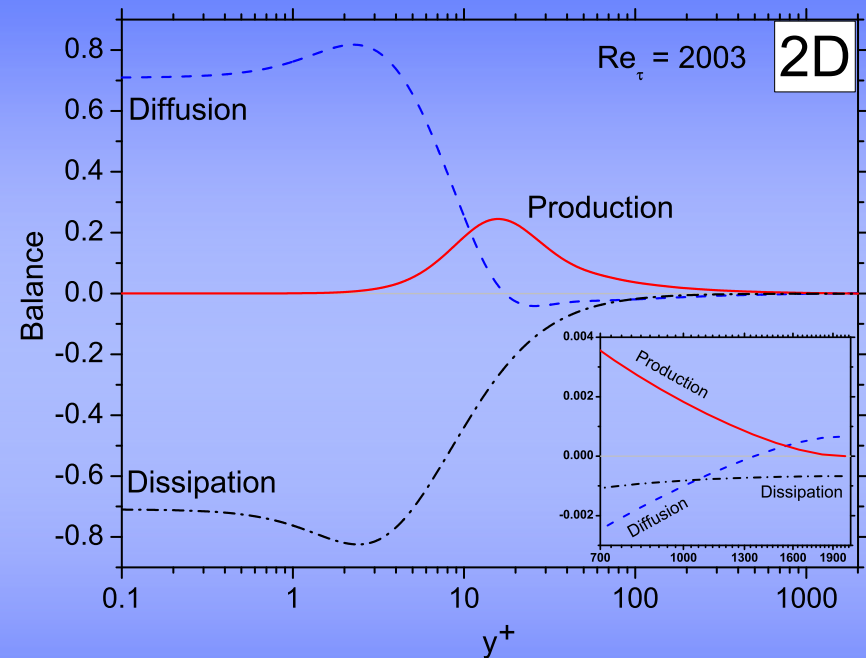
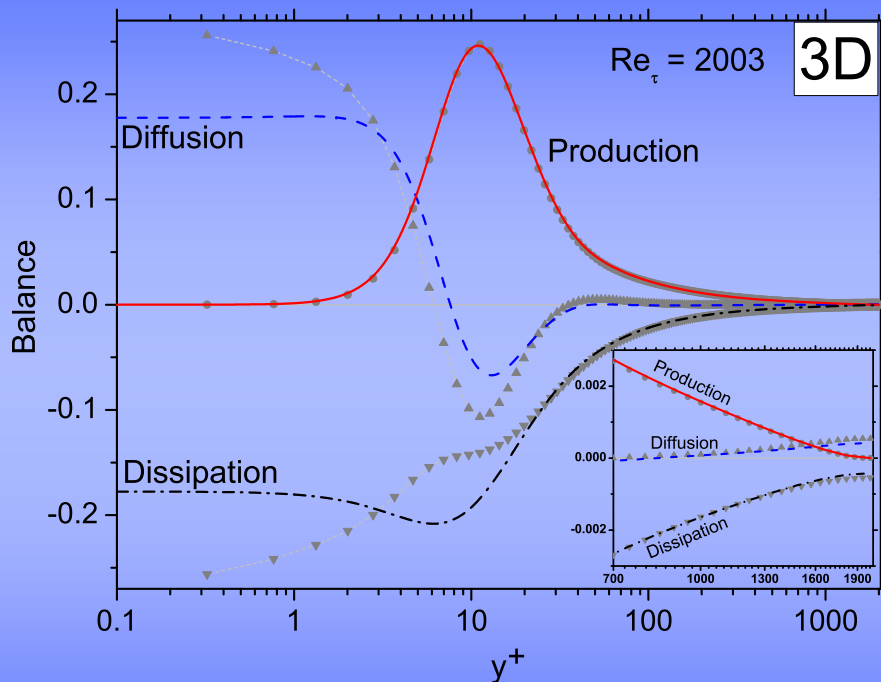


Symbols: Direct Numerical Simulations



# Kinetic Energy Balance

$$P^+(y^+) = \varepsilon^+(y^+) + D^+(y^+), \quad y^+ = y/\ell_\tau = yv/u_\tau.$$



Symbols: Direct Numerical Simulations by

S. Hoyas and J. Jimenez, *Phys. Fluids* **18**, 011702 (2006);

R. G. Moser, J. Kim, and N. N. Mansour, *Phys. Fluids* **11**, 943 (1999).

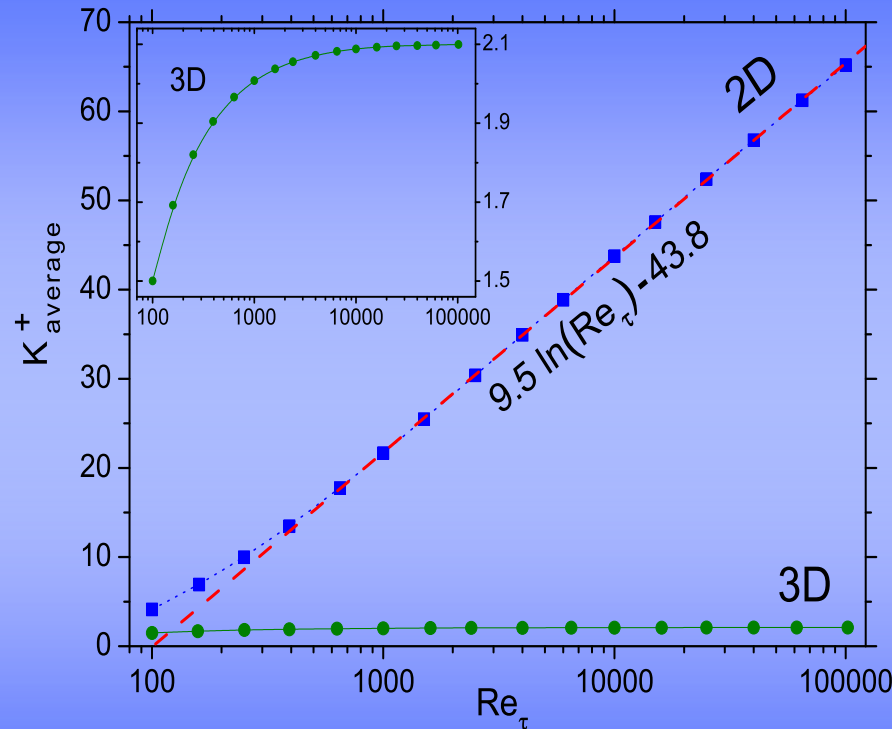
# Conclusions

- Model for 2D and 3D turbulent channel flows: profiles of  $V(y)$ ,  $W(y)$ ,  $K(y)$ , as well as  $S(y)$ ,  $D(y)$ ,  $\varepsilon(y)$ , TKE balance, etc.;
- Direct *energy* cascade (3D) & direct *enstrophy* cascade (2D): difference in the dissipation of  $K(y)$ ;
- Reynolds stress profiles,  $W(y)$ , in 2D and 3D look similar;
- Von-Kármán log-law: “exists” in 3D, just apparent in 2D;
- 2D channel is much energetic, with  $K \sim \ln(\text{Re}_\tau)$ .



# Kinetic Energy vs. $Re_\tau$

$$K^+ = K/u_\tau^2, \quad Re_\tau = Lu_\tau/\nu.$$



$$K_{\text{average}} \equiv \frac{1}{L} \int_0^L K(y) dy \sim u_\tau^2 \ln(Re_\tau).$$





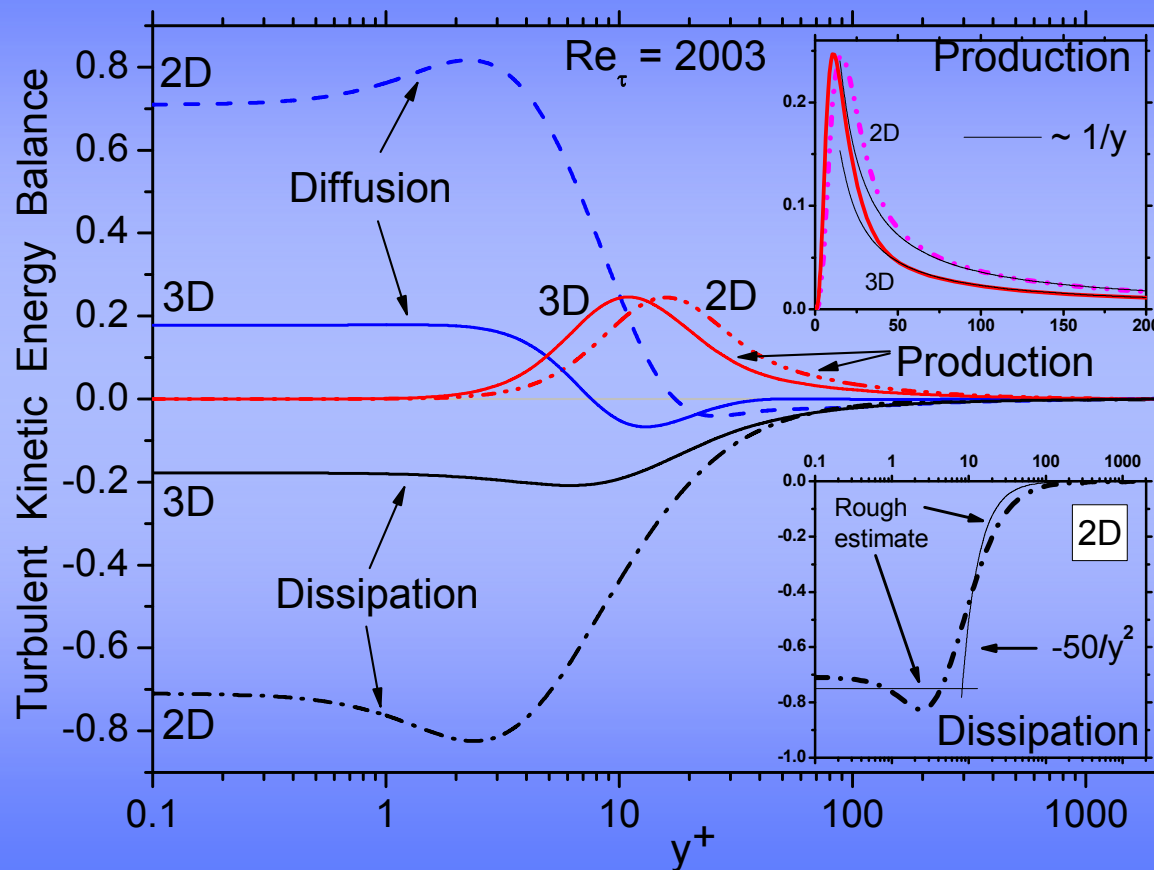
**THANK YOU VERY MUCH!**

**THE END.**



# Appendix: TKE Balance

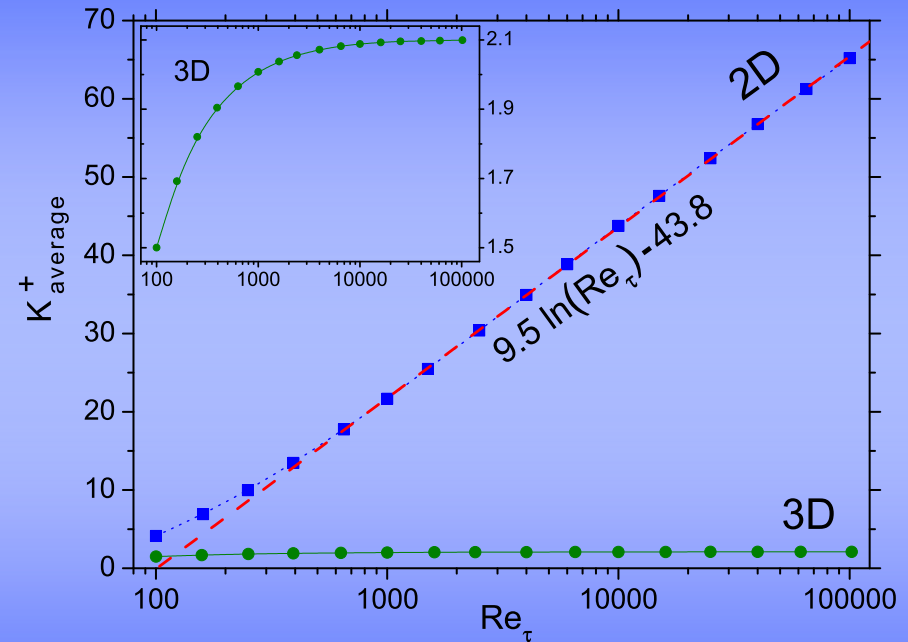
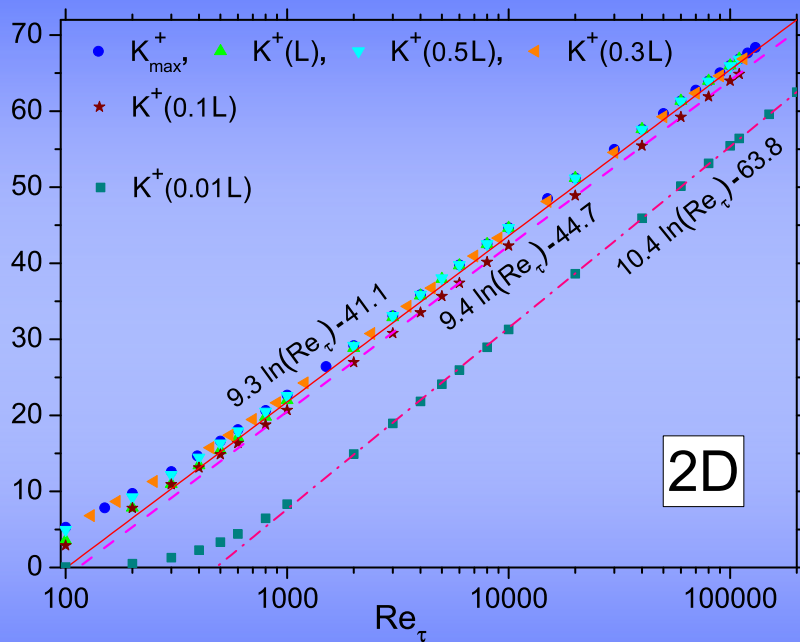
$$P^+(y^+) = \varepsilon^+(y^+) + D^+(y^+), \quad y^+ = y/\ell_\tau = yv/u_\tau.$$





# Kinetic Energy vs. $Re_\tau$

$$K^+ = K/u_\tau^2, \quad Re_\tau = Lu_\tau/\nu.$$



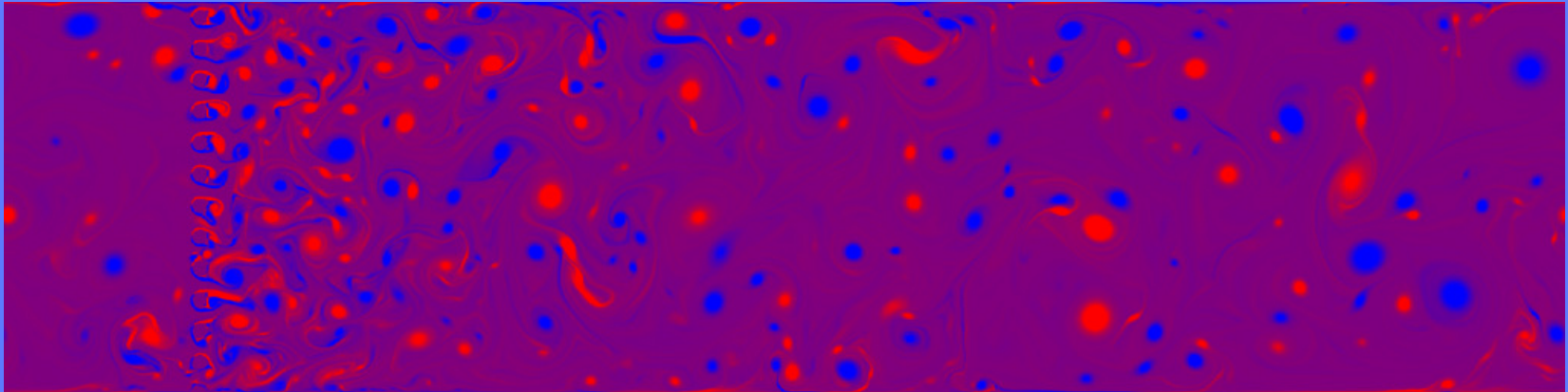
$$K_{\text{typical}} \sim \ln(Re_\tau) + \text{const.}$$

$$K_{\text{average}} \equiv \frac{1}{L} \int_0^L K(y) dy.$$

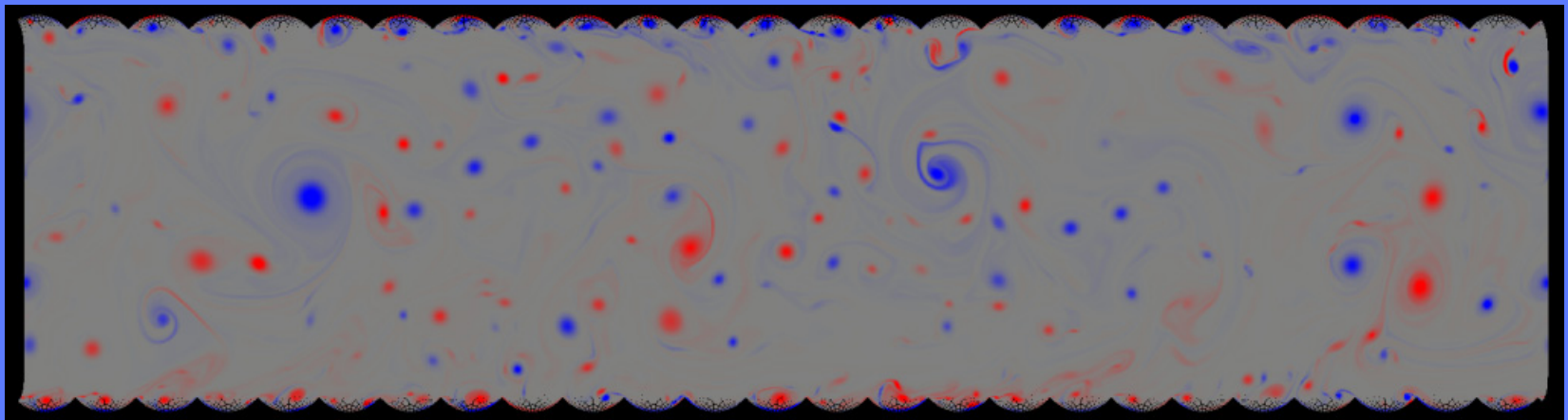


# DNS by Nicholas Guttenberg

Nicholas Guttenberg and Nigel Goldenfeld, *Phys. Rev. E* **79**, 065306(R) (2009):



Grid generated turbulence in a **smooth** pipe. The color scheme indicates positive (blue) and negative (red) vorticity.



Roughness generated turbulence in a **rough** pipe generated via conformal mapping.